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Effect of different fuel NO models on the prediction of NO formation/reduction characteristics in a pulverized coal combustion field

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coal combustion; numerical simulation; NOx; NO formation model; NO reduction model

Abstract

To investigate the effects of fuel NO formation models on the prediction of NO concentrations in a coal combustion field, numerical simulations for a coal combustion field in a 760 kW test furnace were performed. Three models, those proposed by De Soete, Chen et al. and Mitchell et al. were employed to calculate fuel NO formation originating from volatile matter. The results show that the model proposed by Mitchell et al. reproduces the tendency of the experimental data better than the other two models. In addition, the difference between the NO conversion ratios of bituminous coal and sub-bituminous coal that contains a high level of moisture was examined in detail using simulation results from the model of Mitchell et al. It was found that the formation of a region with a low oxygen mole fraction immediately downstream of a region with a high NO production rate is essential to realize a low NO conversion ratio.

1. Introduction

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Coal is an important energy resource from the viewpoint of energy security, because coal quarries and mines are prevalent in various regions all over the world and the number of coal deposits is greater than those of other fossil fuels. For electricity production, a large proportion of coal is consumed by coal-fired thermal power plants that employ the method of pulverized coal combustion. Therefore, emissions from pulverized-coal-fired thermal power plants represent a very important issue for the electric power industry. Although combustion technologies for reducing NOx emissions, such as staged combustion, have been utilized in pulverized-coal-fired plants [1], there is even greater potential for NOx emission reduction. NOx emissions from furnaces can be affected by fuel properties, boiler and burner design, and operating conditions. Recently, demand for utilizing low-grade coals or biomass fuels has increased due to factors such as the increased price of coal and global warming. Thus, the ability to predict changes in NOx emissions caused by utilizing these fuels in large scale boilers is required. Some experimental research has been conducted on the fundamental phenomena of NOx production during coal combustion [e.g., 2-4]. However, the prediction of NOx emissions in actual large-scale boilers requires an understanding of gas flow patterns, temperature, and gas-species concentrations. Numerical simulations of pulverized coal combustion fields are useful to archive such understanding [e.g., 5-28]. Recently, numerical simulations of actual large-scale boilers have been conducted by some researchers [e.g., 7-10]. Numerical simulations for predicting NOx formation characteristics in coal combustion fields have also been conducted [e.g., 11-28]. Since numerical simulation of pulverized coal combustion fields is difficult due to the complexity of coal combustion phenomena, relatively simple models are generally employed. Particularly when simulating a combustion field in a large scale boiler, simplified models are mostly employed due to computational limitations even though there exist many precise models using large computational resources. Therefore, in common with other models, simplified NOx formation models are commonly used to predict NOx concentrations in large-scale boilers by numerical simulation of coal combustion fields. To simulate NOx concentrations in coal combustion fields, the calculation of various NOx formation mechanisms, i.e., the Zeldovich, prompt, and fuel NOx mechanisms, is required. Among these, fuel NOx mechanisms are the most important in coal combustion fields because large portions of the NOx emitted from furnace exits originate from fuel NOx mechanisms [29]. The fuel NOx mechanisms are divided into those for which NOx originates from volatile matter and those for which it originates from char particles. For numerical simulations of coal combustion fields in large-scale boilers, the simple NOx

formation model proposed by De Soete [30] is commonly employed for NOx originates from volatile matter [e.g., 21-28]. However, there are other simple NOx formation models, for instance, Chen et al. [31] made corrections of the reaction parameters of De Soete's model based on the experimental data and recommended the use of proposed parameters instead of the original parameters. Mitchell et al. [32] proposed a simple NOx formation model that employed ammonia as an intermediate spices. Use of the models proposed by Chen et al. and Mitchell et al. for numerical simulation of coal combustion fields is expected to improve the accuracy of NOx emission prediction. However, investigations of the differences in NOx formation characteristics between simulations employing these fuel NOx formation models have not yet been reported.

In this study, numerical simulations for the coal combustion field in a 760 kW test furnace were performed using three models for the formation of fuel NOx originating from volatile matter, i.e., the models proposed by De Soete [30], Chen et al. [31] and Mitchell et al. [32]. The NOx formation characteristics between the combustion fields were compared in detail. In addition, differences in the NO emission characteristics between the combustion fields

of bituminous coal and sub-bituminous coal that contains a high level of moisture are discussed.

2. Numerical simulation

2.1. NOx formation model

In this study, only the production of NO was taken into account because the NOx emitted from an atmospheric pulverized coal combustion field consists mostly of NO, with much lower concentrations of NO₂ and N₂O. In this study, the NO formation models were employed in a "post-processing" fashion, in which a converged combustion flow field solution is obtained before performing the NO prediction. To predict the NO concentration in a coal combustion field, the consideration of four main NO formation mechanisms, i.e., the extended Zeldovich NO mechanism, the prompt NO mechanism, the fuel NO mechanism originating from volatile matter, and the fuel NO mechanism originating from char particles, is required. In this study, three NO formation models were employed and the NO formation characteristics predicted by the three models were compared. Table 1 shows a summary of the three models. For the fuel NO formation mechanism originating from volatile matter, the models proposed by De Soete [30] (Model 1), Chen et al. [31] (Model 2) and Mitchell et al. [32] (Model 3), were employed. For the other NO formation mechanisms, i.e., the Zeldovich NO mechanism, the prompt NO mechanism, and the fuel NO formation mechanism originating from char particles, the same equations were used for all three models. The N

partition ratio between volatile-N and char-N was determined by the modified-TDP model [43] based on the devolatilization database made by the FLASHCHAIN model [52, 53]. The same value of the N partition ratio derived from the modified TDP model was used for all fuel NO formation models listed in Table 1. Detailed explanations of each NO mechanism are described in the following sections.

2.1.1. Extended Zeldovich NO model

To calculate NO production by using the extended Zeldovich NO mechanism, the following formulae are employed:

$$66 N_2 + O \underset{K_{-1}}{\longleftrightarrow} NO + N, (2.1)$$

$$67 N + O_2 \underset{K_{-2}}{\longleftrightarrow} NO + O, (2.2)$$

$$N + OH \underset{K_{-1}}{\overset{K_3}{\longleftrightarrow}} NO + H, \qquad (2.3)$$

where K_i denotes various reaction rate parameters. By employing the quasi-steady-state approximation for the N radical, the NO production rate can be expressed by

$$\frac{d[NO]}{dt} = \frac{\rho_g^2}{1 + \left(K_{-1} \frac{Y_{NO}}{M_{NO}}\right) / \left(K_2 \frac{Y_{O_2}}{M_{O_2}} + K_3 \frac{Y_{OH}}{M_{OH}}\right)} \times \left[2K_1 \frac{Y_O}{M_O} \frac{Y_{N_2}}{M_{N_2}} - \frac{2K_{-1} \frac{Y_{NO}}{M_{NO}}}{K_2 \frac{Y_{O_2}}{M_{O_2}} + K_3 \frac{Y_{OH}}{M_{OH}}} \left(K_{-2} \frac{Y_O}{M_O} \frac{Y_{NO}}{M_{NO}} + K_{-3} \frac{Y_H}{M_H} \frac{Y_{NO}}{M_{NO}}\right)\right], \quad (2.4)$$

where Y_i and M_i denote the mass fraction and molecular weight of species i, respectively. In this study, the following reaction rate parameters, proposed by Baulch et al. [33], are employed for K_i in the above formula:

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$$K_1 = 1.8 \times 10^{11} \exp\left(\frac{-38370}{T_g}\right),$$
 (2.5)

79
$$K_{-1} = 3.8 \times 10^{10} \exp\left(\frac{-425}{T_{o}}\right),$$
 (2.6)

80
$$K_2 = 1.8 \times 10^7 \exp\left(\frac{-4680}{T_g}\right),$$
 (2.7)

81
$$K_{-2} = 3.8 \times 10^6 \exp\left(\frac{-20820}{T_o}\right),$$
 (2.8)

82
$$K_3 = 7.1 \times 10^{10} \exp\left(\frac{-450}{T_g}\right),$$
 (2.9)

83
$$K_{-3} = 1.7 \times 10^{11} \exp\left(\frac{-24560}{T_g}\right),$$
 (2.10)

85 In this study, the effects of OH and H radicals in Eq. (2.4) were ignored. The following equation, proposed by

Westenberg [34], was employed to calculate the mass fraction of the O radical in Eq. (2.4):

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$$Y_{O} = \frac{M_{O}}{\rho_{g}} \left(\rho_{g} \frac{Y_{O_{2}}}{M_{O_{2}}} \right)^{0.5} \frac{3.97 \times 10^{5}}{T_{g}^{0.5}} \exp \left(-\frac{31090}{T_{g}} \right). \tag{2.11}$$

2.1.2. Prompt NO model

To calculate NO production by the prompt NO mechanism, the following formula, proposed by De Soete [30],

92 was employed:

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$$\frac{d[NO]}{dt} = 1.2 \times 10^7 X_{O_2}^b X_{N_2} X_{fiuel} \exp\left(\frac{-2.512 \times 10^5}{RT_g}\right) \frac{P_g}{RT_g}, \qquad (2.12)$$

96 where X_i denotes the mole fraction of species i and b is expressed by the following formulae:

98
$$b = 1.0, (X_{o.} < 4.1 \times 10^{-3}),$$
 (2.13)

$$b = -3.95 - 0.9 \ln X_{o_2}, \left(4.1 \times 10^{-3} \le X_{o_2} < 1.11 \times 10^{-2}\right), \tag{2.14}$$

100
$$b = -0.35 - 0.1 \ln X_{O_2}, (1.11 \times 10^{-2} \le X_{O_2} < 3.0 \times 10^{-2}),$$
 (2.15)

101
$$b = 0, (3.0 \times 10^{-2} \le X_{O_2}).$$
 (2.16)

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2.1.3. Models for NO originating from volatile matter

NO production originating from volatile matter is considered to be very important in pulverized coal combustion fields. For NO production originating from volatile matter, the model proposed by De Soete [30] is commonly employed in numerical simulations of coal combustion fields [21-28]. However, in the detailed chemical reaction simulation conducted by Pederson et al. [35], is was found that the overall reaction rate in the model proposed by De Soete tends to underestimate NO concentration in a low oxygen concentration environment. Therefore, in this study, three models for fuel NO production originating from volatile matter, i.e., the models proposed by De Soete [30] (Model 1), Chen et al. [31] (Model 2) and Mitchell et al. [32] (Model 3), were employed. The differences between the NO formation characteristics of these models were compared in detail.

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2.1.3.1 De Soete's model (Model 1)

114 Figure 1 (a) shows a schematic of the model proposed by De Soete [30]. In this model, HCN evolved from coal particles as volatile matter is consumed by two competing reactions: one producing NO by reaction with O₂ and the other producing N₂ by reaction with NO. The reaction rates can be expressed by the following equations:

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$$\frac{d[HCN]}{dt} = -1.0 \times 10^{10} X_{HCN} X_{O_2}^b \exp\left(\frac{-2.805 \times 10^5}{RT_g}\right) \frac{P_g}{RT_g}, \qquad (2.17)$$

119
$$\frac{d[HCN]}{dt} = -3.0 \times 10^{12} X_{HCN} X_{NO} \exp\left(\frac{-2.512 \times 10^5}{RT_g}\right) \frac{P_g}{RT_g}, \qquad (2.18)$$

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where b can be calculated by using Eqs. (2.13) - (2.16).

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2.1.3.2 Chen et al.'s model (Model 2)

124 Chen et al. conducted chemical reaction simulations with elementary kinetics and compared simulation results with 125 experimental data [31]. They found that employing the following equations, instead of the original equations from 126 De Soete, provided a better prediction:

128
$$\frac{d[HCN]}{dt} = -1.5 \times 10^{10} X_{HCN} X_{O_2}^b \exp\left(\frac{-2.646 \times 10^5}{RT_g}\right) \frac{P_g}{RT_g}, \qquad (2.19)$$

129
$$\frac{d[HCN]}{dt} = -1.1 \times 10^{12} X_{HCN} X_{NO} \exp\left(\frac{-2.454 \times 10^5}{RT_g}\right) \frac{P_g}{RT_g}.$$
 (2.20)

2.1.3.3 Mitchell et al.'s model (Model 3)

- Figure 1 (b) shows a schematic of the model proposed by Mitchell et al. [32]. In this model, HCN is converted to
- 133 NH₃, followed by the reactions of NH₃ to NO and N₂. The reaction rates are expressed by the following equations:

135
$$\frac{d[HCN]}{dt} = -1.94 \times 10^{15} X_{HCN} X_{O_2} \exp\left(\frac{-3.282 \times 10^5}{RT_g}\right) \frac{P_g}{RT_g}, \qquad (2.21)$$

136
$$\frac{d[NH_3]}{dt} = \frac{-3.48 \times 10^{20} X_{NH_3} X_{O_2} \exp(-4.186 \times 10^5 / RT_g)}{1 + 6.90 \times 10^{-6} X_{O_3} \exp(1.758 \times 10^5 / RT_g)} \frac{P_g}{RT_g},$$
 (2.22)

137
$$\frac{d[NH_3]}{dt} = -6.22 \times 10^{14} X_{NH_3} X_{NO} \exp\left(\frac{-2.302 \times 10^5}{RT_g}\right) \frac{P_g}{RT_g},$$
 (2.23)

138
$$\frac{d[HCN]}{dt} = 1.00 \times 10^4 X_{NO} X_{CmHb} \frac{P_g}{RT_g}, \qquad (2.24)$$

where X_{CmHb} in Eq. (2.24) is the mole fraction of gaseous hydrocarbons, which is represented by CH₃ [36]. In this study, the concentration of gaseous hydrocarbons could not be calculated in detail because the volatile matter was treated as a mixture of postulated substances; therefore, its detailed reaction chemistry of the volatile matter could not be effectively quantified. However, it was considered that the concentration of gaseous hydrocarbons is correlated with that of volatile matter. Therefore, the concentration of hydrocarbons was assumed to be proportional to that of volatile matter according to the following formula:

$$147 X_{C_mH_b} = C_1 X_{VM}, (2.25)$$

- where X_{VM} is the mole fraction of volatile matter, and C_I is a constant. The value of C_I was set to 1×10^{-2} in this study.
- The effect of the value of C_l on NO formation is discussed in the section 3.1.

152 2.1.4. Model for NO production originating from char particles

To calculate NO production originating from char particles, the following formula, proposed by Lockwood et al.

154 [37], was employed:

$$S_{NO,Ch} = -\frac{dC}{dt} m_{N,Ch} F_c \eta_{N,Char}, \qquad (2.26)$$

where $\eta_{N,Char}$ is the conversion factor of the nitrogen in the char to NO; it was set to 0.2 in this study.

2.1.5. Model for NO reduction by char particles

- The NO-char reaction has been investigated by various researchers [38-41] and is believed to have a large effect on
- NO reduction in a pulverized coal combustion environment. The rate of NO reduction by char can be calculated by
- the following equation:

$$\frac{d[NO]}{dt} = -A_{re}X_{NO} \exp\left(-\frac{E_{re}}{RT_p}\right)S_p \frac{P_g}{RT_g}, \qquad (2.27)$$

- where E_{re} was set to 1.33 × 10⁶ J/mol (after Aarna et al. [38], who obtained this value by averaging the data from
- various studies). S_p is the external surface area of char particles per unit volume (m^2/m^3). The pre exponential factor,
- A_{re} , is was to 2.0×10^4 m/s in this study.

2.2. Comparison of NO production/reduction rates between the three volatile fuel NO models

Before conducting the numerical simulation, the NO production/reduction rates of the three volatile fuel NO models as functions of the O₂ mole fraction, the NO mole fraction, and the gas temperature were examined, as shown in Fig. 2. The basic conditions for the calculations were: HCN concentration, 3000 ppm; volatile matter concentration (VM), 10%; NO concentration, 100 ppm; and gas temperature, 1800 K. As can be understood from Eqs. (2.17) – (2.24) and Fig. 1, the volatile fuel NO mechanisms have both production and reduction rates. The NH₃ concentration is required to calculate the NO production and reduction rates in Model 3 (Mitchell et al.), whereas it is not required for Models 1 and 2. However, the NH₃ consumption rates for production and reduction (Eqs. (2.22) and (2.23)) are much higher than the consumption rate of HCN (Eq. (2.21)). Consequently, the concentration of NH₃ is much lower than that of HCN [35]. Considering this fact, the steady-state approximation [42] for NH₃ was

adopted in this calculation, i.e., it was assumed that the overall NO production and reduction rates were controlled by the HCN consumption rate (Eq. (2.21)). This steady-state approximation was only adapted to the calculation for Fig. 2, and was not adapted to the numerical simulation for the coal combustion field in the 760 kW test furnace. The ratio of the NO production rate to the NO reduction rate was calculated by using Eqs. (2.22) and (2.23) with same value of NH₃ concentration. In Fig. 2, the solid and dotted lines indicate the production and reduction rates, respectively, for Model 1 (blue), Model 2 (green), and Model 3 (red). If the solid line is higher than the dotted line, the net NO production rate is positive. Conversely, if the dotted line is higher than the solid line, the net NO production rate is negative. In Figs. 2 (a) and 2 (b), the NO production/reduction rates as a function of the O₂ mole fraction at different gas temperatures are compared. The net NO production rate increases with increasing O2 mole fraction and switches from negative to positive at a certain O₂ mole fraction (intersection point of the solid and dotted lines) for all three models. Throughout the range of O₂ mole fractions in Figs. 2 (a) and 2 (b), the order of the net NO production rate is Model 1 < Model 2 < Model 3. However, there is no significant difference between the models in terms of the overall trend. In Figs. 2 (c) and 2 (d), the NO production/reduction rates as a function of the NO mole fraction at different O2 concentrations are compared. For all three models, the NO reduction rate increases with increasing NO mole fraction. As can be understood from Eqs. (2.17) and (2.19), the NO production rate for Models 1 and 2 are unchanged with NO mole fraction, whereas the NO production rate decreases with increasing NO mole fraction. However, there is no significant difference between the models in terms of the overall trend. Throughout the range of NO mole fractions in Figs. 2 (c) and 2 (d), the order of the net NO production rate is Model 1 < Model 2 < Model 3. Figures 2 (e) and 2 (f) show the NO production/reduction rates as functions of gas temperature at different O2 concentrations. There is a significant difference between Model 3 and Models 1 and 2 in terms of the overall trend. There is no significant difference between the shapes of the curves of NO production and reduction rates for Models 1 and 2. Consequently, the NO production and reduction curves do not intersect for Models 1 and 2 in Fig. 2 (f), i.e., the net NO production rates for Models 1 and 2 are negative throughout the gas temperature range in Fig 2 (f). On the other hand, the shapes of the curves of NO production and reduction rates for Model 3 are totally different. Consequently, there is an obvious intersection point between the NO production and reduction rate curves for Model 3 in Fig. 2 (f), i.e., the net NO production rate switches from negative to positive at around 1350 K with the increase in gas temperature. This large difference between the models in terms of the overall trend with gas temperature change causes the significant difference in NO concentration distribution in the furnace,

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as discussed later. The order of the net NO production rate at low gas temperature is Model 3 < Model 1 < Model 2, whereas that at high gas temperature it is Model 1 < Model 2 < Model 3.

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2.3. Numerical methods

- Other than the NO formation model, the models used in this study for numerical simulation of the pulverized coal combustion field were the same as those of Hashimoto et al. [43]. The models for the main phenomena are
- summarized in Table 2.
- The gas-phase time-averaged continuity equation and conservation equations of momentum, turbulent kinetic
- energy, dissipation, enthalpy and species are

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$$\frac{\partial}{\partial x_i} \left(\rho_g u_i \right) = S_c \,, \tag{2.28}$$

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$$\frac{\partial}{\partial x_i} \left(\rho_g u_i \phi \right) = \frac{\partial}{\partial x_i} \left(\Gamma_{\phi} \frac{\partial \phi}{\partial x_i} \right) + S_{\phi} + S_{p\phi} , \qquad (2.29)$$

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- where S_c denotes the mass source term from the representative particles to the gas phase. ϕ denotes the generalized variables expressing the fluid velocity components u_i , the turbulent kinetic energy k, the rate of eddy dissipation ε , the fluid enthalpy h and the mass fraction of chemical species Y_i . Γ_{ϕ} denotes the turbulent exchange coefficient, and S_f and $S_{p\phi}$ represent the gas-phase source terms that exist in addition to the convection and diffusion terms and the particle-phase source terms, respectively. The continuity and momentum equations were solved using the pressure-implicit with splitting of operators (PISO) algorithm [50].
 - The mass of the representative coal particles m_p was calculated using the following equation by considering the evaporation of moisture, the evolution of volatile matter by devolatilization and the evolution of carbon monoxide by the char combustion from the particles:

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$$\frac{dm_p}{dt} = \frac{dm_{moist}}{dt} + \frac{dm_{vola}}{dt} + \frac{dm_{char}}{dt}$$
 (2.30)

- where, m_{moist} , m_{vola} and m_{char} are the mass of moisture, volatile matter and fixed carbon in the particle, respectively.
- The decrease rates of m_{moist} , m_{vola} and m_{char} were calculated by the modified TDP model [43] and the char

- combustion model proposed by Field et al. [47] with the char combustion zone transition proposed by Essenhigh et
- 237 al. [47], as listed in Table 2.
- The equation of motion for the representative coal particles is given by

240
$$m_{p} \frac{du_{pi}}{dt} = \frac{1}{2} C_{d} \rho_{p} A_{p} | u_{fj} - u_{pj} | (u_{fi} - u_{pi}),$$
 (2.31)

241
$$C_d = 24(1 + 0.15 \text{Re}_p^{0.687})/\text{Re}_p$$
, (2.32)

242 Re_p =
$$D_p | u_{fi} - u_{pi} | / v$$
. (2.33)

243

- The particle temperature T_p was calculated using the following equation by considering the heat transfer due to
- convection, radiation, heat loss due to the evaporation of moisture and the devolatilization reaction in the coal
- particle, and heat gain due to char combustion:

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$$m_{p}c_{p,p}\frac{dT_{p}}{dt} = -A_{s}h(T_{p} - T_{g}) + A_{s}\varepsilon_{p}\sigma(\Theta_{R}^{4} - T_{p}^{4}) + \Delta h_{lat}\frac{dm_{p,w}}{dt} + \Delta h_{dev}\frac{dm_{p,v}}{dt} + \dot{q}_{char}, \quad (2.34)$$

249
$$h = k_g Z (2 + 0.6 \text{Re}_p^{1/2} \text{Pr}^{1/3}) / (e^Z - 1) D_p$$
, (2.35)

250
$$Z = -c_{p,g} \left(\frac{dm_p}{dt} \right) / \pi D_p k_g \left(2 + 0.6 \operatorname{Re}_p^{1/2} \operatorname{Pr}^{1/3} \right),$$
 (2.36)

 $\Theta_{R} = (I/4\sigma)^{1/4}. \tag{2.37}$

- 253 The absorptivities of the coal particles and wall were assumed to be 0.85 and 0.4, respectively. Also, the absorption
- coefficient of the gas was set at 0.075. The interaction of the conserved properties between the gas phase and the
- coal particles was calculated by the particle-source-in cell (PSI-Cell) technique [51].
- In this study, the modified tabulated-devolatilization-process (TDP) model [43] was employed for devolatilization
- of the coal particle. The FLASHCHAIN model [52, 53] was used to produce the devolatilization database for the
- TDP model.
- Gaseous combustion between the volatile matter and air was calculated using a combined model of the kinetics and
- 260 eddy dissipation models [46]. The chemical mechanism consists of the following global reactions:

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$$VM + \alpha O_2 \rightarrow \beta CO + \gamma H_2 O, \qquad (2.38)$$

$$263 \qquad CO + O_2 \rightarrow CO_2. \tag{2.39}$$

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In this study, the parameters proposed by Westbrook et al. [54] for C_7H_8 oxidation were employed for the rate parameters regarding the kinetics in Eq. (2.38). The parameters proposed by Hautman et al. [55] for CO oxidation were employed for the rate parameters regarding the kinetics in Eq. (2.39).

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2.4. Computational domain and conditions

The test furnace studied here is located at the Energy Engineering Research Laboratory of CRIEPI, in which an advanced low-NOx burner (CI-α burner [3]) with a coal combustion capacity of about 100 kg/h is installed. The furnace is a water-cooled furnace made of steel with refractory materials placed on the inside wall. The diameter and length of the furnace are 0.85 m and 8 m, respectively. The configuration of the computational domain is shown in Fig. 3. The domain was designed to match the actual configuration accurately. The computational domain was half of the furnace, and a periodic condition was applied in the azimuthal direction. Combustion air was injected into the furnace through the burner and staged combustion air ports located 3.0 m from the burner outlet. The air passing through the burner was divided into primary, secondary, and tertiary air. The primary air, which carries pulverized coal, had straight motion. The secondary and tertiary air had strong swirling motions. The swirl vane angles for the secondary and tertiary air were set to 81 deg. and 63 deg., respectively, which are the optimum values for bituminous coal (these values are zero when the swirl force is zero). The operating conditions of the furnace in the simulation were set to correspond with those in our experiment [56]. The thermal input of the coal combustion test furnace was 760 kW (the feed rate for bituminous coal was approximately 100 kg/h). The excess air ratio was 1.24, and the O₂ mole fraction at the furnace outlet was 4.0%. The staged combustion air ratio was set to 30%. The mass ratio of the pulverized coal (dry base) to the primary air was 1:2.2, and the mass ratio of secondary air to tertiary air was 1:6. The temperature of the primary air was set to 353 K, and that of the secondary and tertiary air was 598 K. Regarding the boundary condition on the wall, the temperature outside the furnace was assumed to be 308 K and thermal resistance was set to 0.04 (m² s K)/J.

The properties of the coals are listed in Table 3. The size distributions of coal particles used in the experiment [56]

are shown in the Fig. 4. It was assumed that the pulverized coal consisted of particles with initial diameters of 5, 20, 40, 60, 80, and 100 μm. The mass fractions of the coal particles were set to correspond with the actual particle size distribution in the experiment.

Four cases with different sub-bituminous coal mixing ratios were performed in this study, as listed in Table 4. Numerical simulations with sub-bituminous coal mixing ratios of 0%, 25%, 75% and 100% were conducted. The total thermal input was set at 760 kW for all cases, as described above. For each case, the calculations listed in Table 1 were conducted for the three models. Therefore, a total of 12 calculations were conducted.

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3. Results and discussion

3.1. Effect of fuel NO model

Figure 5 shows the simulation results of the gas velocity vectors, gas temperatures and O₂ mole fractions for Cases 1 and 4. For the gas velocity vectors, the colors in (a-1) and (a-2) indicate axial velocity. Also shown are the distributions of the gas temperature measured by the thermocouple and O2 mole fraction measured during the experiments by a gas analyzer with a suction probe [56]. The radiative heat loss from the thermocouple was not corrected because of the difficulty of estimating the furnace wall temperature. The tendency of the simulation results is consistent with that of the experimental results. For instance, due to the large flame lift-off, the temperatures near the burner exit for both the simulation and experimental results in Case 4 (Fig. 5 (b-2) and (c-2)) are markedly lower than that in Case 1 (Fig. 5 (b-1) and (c-1)). Similarly, the oxygen mole fractions near the burner exit for both the simulation and experimental results in Case 4 (Fig. 5 (d-2) and (e-2)) are high, whereas that in Case 1 (Fig. 5 (d-1) and (e-1)) is low. These marked differences between the gas temperatures and oxygen mole fractions in Cases 1 and 4 are due to the large flame lift-off for Case 4, as discussed in detail in Ref. [43]. It should be noted that because of the large flame lift-off for Case 4, there is a large difference between the gas velocity vector fields in Cases 1 and 4. Figure 6 shows the effect of C_I in Eq. (2.25) for Model 3 on NO mole fraction at the furnace exit. The NO mole fraction at the furnace exit decreases with increasing C_l . Even though the absolute predicted values of the NO mole fraction with $C_I = 1$ for Case 1 and 2 agree well with the measured data, the simulation with $C_I = 1$ could not reproduce the increase in NO mole fraction with increasing Wara coal mixing ratio. The simulation with $C_I = 1 \times 10^{-4}$ overestimates NO mole fraction too much. Therefore, $C_I = 1 \times 10^{-2}$ was used in Model 3 in all of the following discussion.

Figure 7 shows the NO mole fraction at the furnace exit as a function of Wara coal mixing ratio. In addition to the simulation results from the three different volatile fuel NO models, the measurement results of the experiment [56] are also shown. The NO mole fraction at the furnace exit of the experiment gradually increases with an increase in the Wara coal mixing ratio. The NO mole fractions for Models 2 and 3 increase with increasing Wara coal mixing ratio, whereas that for Model 1 decreases as this ratio increases. If only Fig. 7 is considered, it may appear that Model 2 yields better results than the other models do. However, this appearance may change if the distribution of the NO mole fraction in the furnace is also considered, as discussed below. Figure 8 shows the distribution of the NO mole fraction on the central axis of the furnace. Model 3 yields the best NO mole fraction of the models. Although overestimations of the NO mole fraction at Z = 0.1 m by Model 3 are observed for (a) Case 1 (100% Newlands) and (b) Case 2 (75% Newlands + 25% Wara), the overall discrepancy for Model 3 is smaller than that of the other models. Models 1 and 2 underestimate the NO mole fraction at the burner zone (from Z = 0 m to 3 m). Figures 9 show the two dimensional distributions of the NO mole fraction for Cases 1 and 4. From the figure, it is evident that Model 3 yields a better NO mole fraction distribution than the other models do. A typical example of this tendency can be observed in the region bounded by the white dotted line in Fig. 9 (a-1), (b-1) and (c-1). In the experimental and simulation results of Model 3, the NO mole fraction in this region is high; whereas for Models 1 and 2, it is low. This significant differences between the NO mole fractions of the different models can be explained by the differences in the volatile fuel NO production/reduction characteristics shown in Fig. 2. In this region, the O₂ mole fraction is low (Fig. 9 (e-1)) and the gas temperature is high (Fig. 9 (f-1)). As can be seen in Figs. 2 (a) and 2 (b), the net NO production rate by the volatile fuel NO mechanism strongly depends on the O2 mole fraction, i.e., the net NO production rate by the volatile fuel NO mechanism decreases with increasing O₂ mole fraction; this overall tendency is the same for all three models. However, among the models, there are significant difference in the overall gas temperature trends, as shown in Figs. 2 (e) and 2 (f). The net NO production for Model 3 can be positive if the temperature is high, even if the O₂ mole fraction is low, as shown in Fig. 2 (f). In contrast, that for Models 1 and 2 cannot be positive. In other words, for Model 1 and 2, whether the net NO production rate is positive or negative is determined almost exclusively by the O₂ mole fraction; whereas for Model 3, it is determined not only by O₂, but also by gas temperature. This difference between the dependences of the net NO production by the volatile fuel NO mechanism on gas temperature is the main cause for the significant differences between the predicted NO

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mole fraction distributions. In the region bounded by the white dotted line in Fig. 9, the O2 mole fraction is low (Fig. 9 (e-1)) and the gas temperature is high (Fig. 9 (f-1)); therefore, significant differences between the models in terms of NO mole fraction were observed in this region. Similarly, significant differences among the models in terms of NO mole fractions in the region bounded by the white dotted line are evident in Fig. 9 (a-2), (b-2) and (c-2). In the experimental and simulation results of Model 3, the NO mole fraction in this region is high; whereas for Models 1 and 2, it is low. The NO mole fraction around the central axis at Z > 0.75 m is low for all of the simulation results, whereas for the experimental results, it is high. However, the simulation results of Model 3 are closer to the experimental results than that of the other models. Figure 10 shows the rates of NO production or reduction by each mechanism on the central axis of the furnace for Case 1. The contributions of the Zeldovich and prompt NO mechanisms to NO production or reduction of NO are quite small. On the other hand, the contribution of volatile fuel NO production is very large compared with that of the other mechanisms. This is why the differences in modeled volatile fuel NO strongly affect the distribution of the NO mole fraction in the furnace. The NO production rate by volatile fuel NO mechanism in Model 3 is greater than that for the other two models. Although the NO reduction rate by volatile fuel NO mechanism in Model 3 is also greater than that for the other two models, the NO mole fraction steeply increases from Z = 0.6 to Z = 0.75 because of the high volatile fuel NO production rate. The rate of NO reduction by char particles in the results of Model 3 is higher than that for the other two models. This is because the NO concentration in the results of Model 3 is higher than that for the other two models. As discussed above, the model proposed by Mitchell et al. can reproduce the tendency of the measurement results better than the other two models. However, there are some quantitative discrepancies between the simulation results of the model by Mitchell et al. and the experimental results, as observed in Figs. 8 and 9. The main reason of the discrepancies is considered to be the overestimation of NO production rate in high oxygen concentration region. For instance, the calculated NO mole fraction near the furnace wall in Fig 9, in which the O2 concentration is higher than that near the central axis, is higher than the measured NO mole fraction. The higher NO production rate in the condition of high oxygen concentration for Model 3 compared to other two models can be also confirmed in Fig. 2 (a) and (b). Further improvement of the models is required to predict the NO production/reduction characteristics

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3.2. Cause of differences between NO conversion ratios of two coals

In Fig. 7, NO emission increases with increasing Wara coal mixing ratio. As previously mentioned, a large portion of the NO emitted from a furnace exit originates from fuel NO mechanisms. Therefore, NO emission depends strongly on the amount of fuel nitrogen fed into the furnace. For this reason, the conversion ratio of fuel nitrogen to NO is widely used as an indicator of the combustion performance of coal against the NO emission. The conversion ratio is defined as follows:

where CR_{NO} , EX_{NO} and F_N are the conversion ratio of fuel nitrogen to NO [%], the NO emission rate at the furnace

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$$\operatorname{CR}_{NO} = \frac{EX_{NO}}{F_N} \times 100$$
, (3.1)

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exit [mol/s], and the fuel nitrogen feeding rate from the burner [mol/s], respectively. Figure 11 shows CR_{NO} as a function of the Wara coal mixing ratio, revealing that CR_{NO} increases with increasing Wara coal mixing ratio. This indicates that the NO formation/reduction characteristics are affected by the Wara coal mixing ratio. In this section, the cause of the increase in CR_{NO} is discussed using the simulation results of Model 3. Figure 12 shows a comparison of the two-dimensional distributions of net NO production/reduction rate, NO mole fraction, gas temperature, and O2 mole fraction for Cases 1 and 4 on the cross section at the center of the furnace. The main gas flow patterns are indicated by the black arrows in the figures. The position of the flame front, which is located at the position where the gas temperature steeply increases, is also indicated by solid white lines. It is found that the position of the flame front for Case 4 is markedly different from that for Case 1. Due to the large flame lift-off, the distance between the flame front and burner exit for Case 4 is larger than that for Case 1. For both cases, a region with a high net NO production rate is accompanied by the flame front (Fig. 12 (a-1) and (a-2)). In Fig. 12 (a-1), for Case 1, there is a region with a high net NO reduction rate immediately downstream of a region with a high net NO production rate. For Case 4, however, there is no region with high net NO reduction rate in downstream of a region with a high net NO production rate, as indicated in Fig. 12 (a-2). Necessary conditions for a high NO reduction rate are a high NO mole fraction, high gas temperature, and low oxygen mole fraction, as can be understood from Eqs. (2.22) and (2.23). The NO reduction rate increases with increases in X_{NO} and T_g in Eq. (2.23). However, when the oxygen mole fraction is high, the net NO production rate is also high because the NO production

rate, expressed by Eq. (2.22), becomes larger than the NO reduction rate, expressed by Eq. (2.23). For Case 1, all of the necessary conditions for a high NO reduction rate are met in the region with high net NO reduction, i.e., the conditions of a high NO mole fraction (Fig. 12 (b-1)), high gas temperature (Fig. 12 (c-1)), and low oxygen mole fraction (Fig. 12 (d-1)) are satisfied in the region bounded by the white dotted line. For Case 4, on the other hand, although the conditions of a high NO mole fraction (Fig. 12 (b-2)) and high gas temperature (Fig. 12 (c-2)) are satisfied, the condition of low oxygen mole fraction is not satisfied in the region bounded by white dotted lines (Fig. 12 (d-2)), which is immediately downstream of the region with a high NO production rate. Whether or not the region with a high NO reduction rate exists is the main cause of the difference between CR_{NO} in Cases 1 and 4, as shown in Fig. 11.

From the above discussion, it was found that the formation of a region with a low oxygen mole fraction immediately downstream of a region with a high net NO production rate is the essential condition for low CR_{NO} . This finding should be considered in the development of low NOx burners and the optimization of operating conditions to reduce NOx emission.

4. Conclusions

conversion ratio.

In this study, numerical simulations of the coal combustion field in a 760 kW test furnace were performed by employing three models for fuel NO formation to investigate the effect of fuel NO formation models on the prediction of NO formation/reduction characteristics. The results show significant differences between the NO concentration predictions of the three models. The model proposed by Mitchell et al. could reproduce the tendency of the measurement results better than the models proposed by De Soete and Chen et al. It was found that significant differences between the NO concentration predictions of the models resulted from differences between dependencies of the net NO production rate by the volatile fuel NO mechanism on the gas temperature in the different models.

Furthermore, the cause of the difference between the NO conversion ratios of bituminous coal and sub-bituminous coal that contains a high level of moisture was investigated using simulation results obtained using the model of Mitchell et al. It was concluded that the formation of a region with a low oxygen mole fraction immediately

 downstream of a region with a high net NO production rate is the essential condition for realizing a low NO

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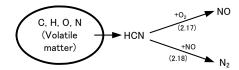
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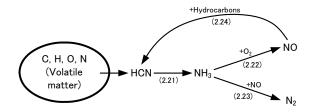
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Figure and table captions

- Fig. 1. Schematics of reaction models for NO originating from volatile matter.
- Fig. 2. Comparison of NO production/reduction rates between three volatile fuel NO models
- Fig. 3. Computational domain.
- Fig. 4. Diameter distribution of coal particles used in the experiment [56].
- Fig. 5. Gas velocity vectors and distributions of gas temperature and O₂ mole fraction for Case 1 (100% Newlands) and Case 4 (100% Wara).
- Fig. 6. Effect of C_1 in Eq. (2.25) for Model 3 on NO mole fraction at furnace exit.
- Fig. 7. NO mole fraction at furnace exit versus Wara coal mixing ratio.
- Fig. 8. Distributions of NO mole fraction on central axis of furnace.
- Fig. 9. Distributions of NO mole fraction for Case 1 (100% Newlands) and Case 4 (100% Wara).
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- Fig. 11. NOx conversion ratio as versus Wara coal mixing ratio.
- Fig. 12. Comparison of distributions of NO production/reduction rate, NO mole fraction, gas temperature and O_2 mole fraction for Case 1 (100% Newlands) and Case 4 (100% Wara).
- Table 1 NO formation/reduction model.
- Table 2 Summary of mathematical models used in simulations.
- Table 3. Coal properties.
- Table 4. Cases investigated in this study.



(a) Model proposed by De Soete.



(b) Model proposed by Mitchell et al.

Fig. 1. Schematics of reaction models for NO originating from volatile matter.

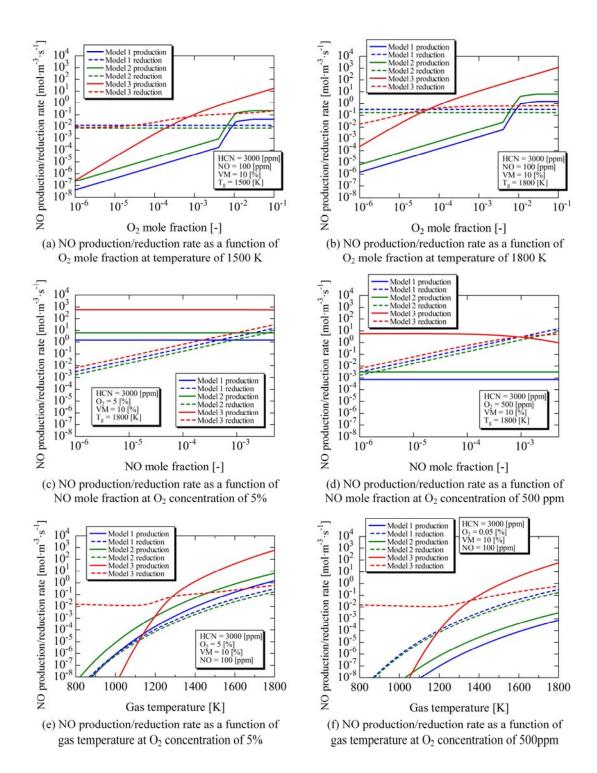


Fig. 2. Comparison of NO production/reduction rates between three volatile fuel NO models.

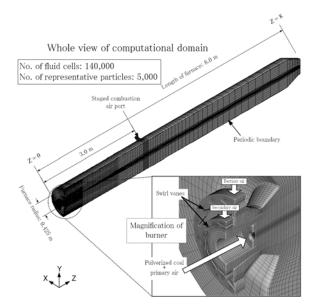


Fig. 3. Computational domain.

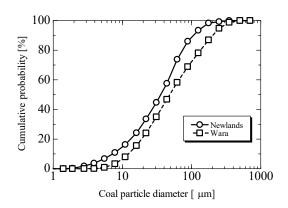


Fig. 4. Coal particle diameter distribution of used

in the experiment [56].

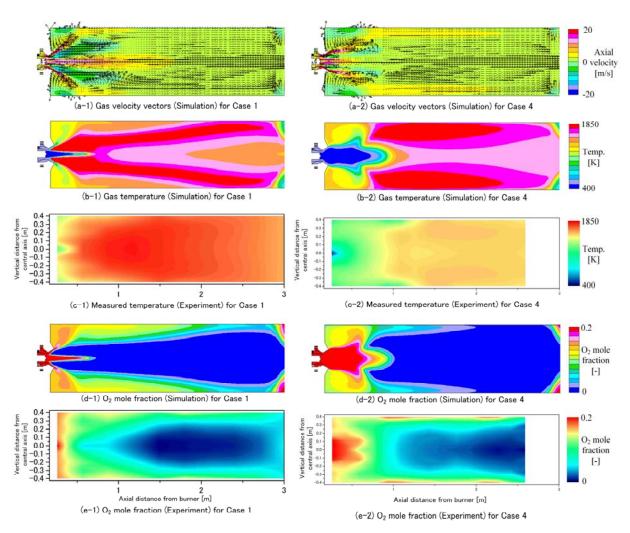


Fig. 5. Gas velocity vectors and distributions of gas temperature and O_2 mole fraction for Case 1 (100% Newlands) and Case 4 (100% Wara).

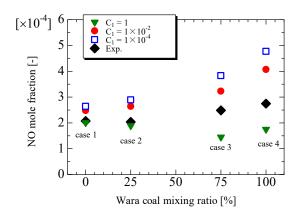


Fig. 6. Effect of C_I in Eq. (2.25) for Model 3 on

NO mole fraction at furnace exit.

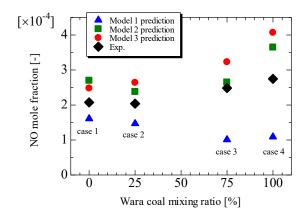


Fig. 7. NO mole fraction at furnace exit versus

Wara coal mixing ratio.

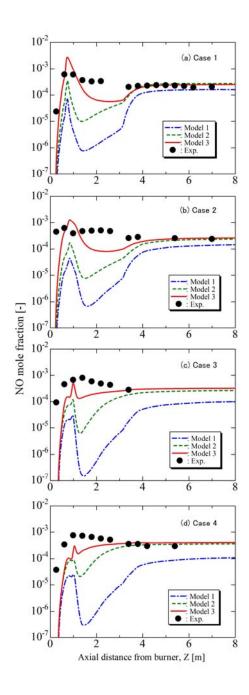


Fig. 8. Distributions of NO mole fraction on central axis of furnace.

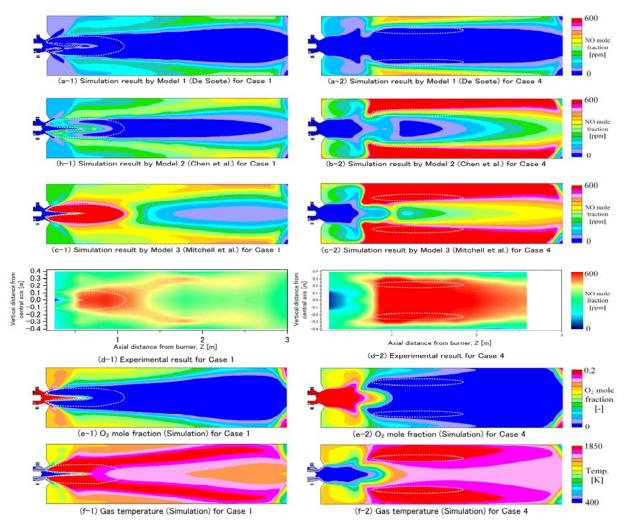


Fig. 9. Distributions of NO mole fraction for Case 1 (100% Newlands) and Case 4 (100% Wara).

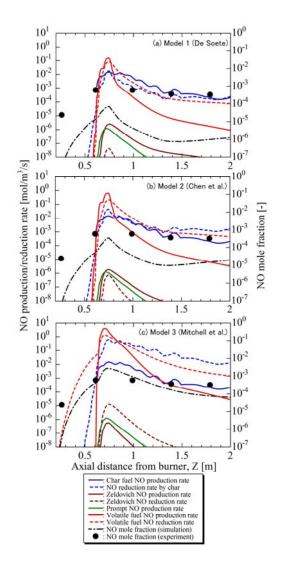


Fig. 10. NO production/reduction rate on central axis of furnace (Case 1).

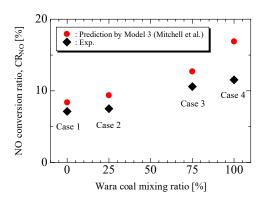


Fig. 11. NOx conversion ratio as versus Wara coal mixing ratio.

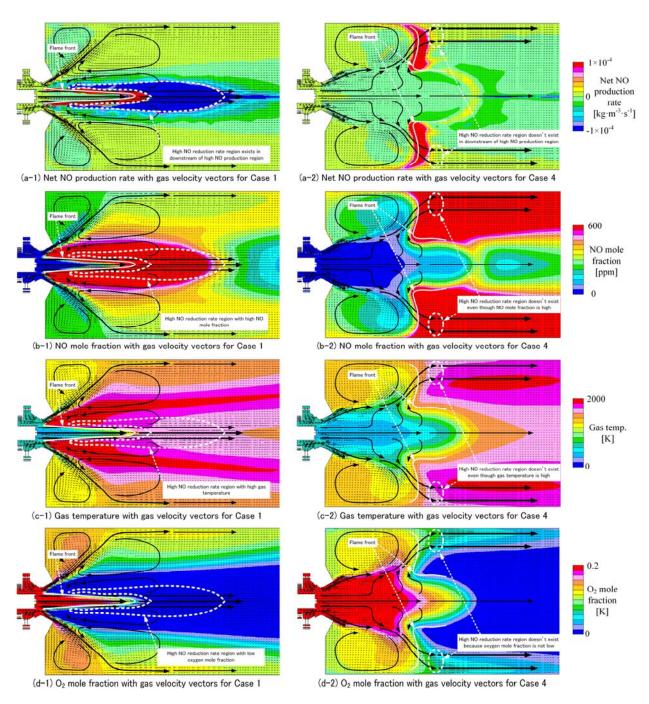


Fig. 12. Comparison of distributions of NO production/reduction rate, NO mole fraction, gas temperature and O₂ mole fraction for Case 1 (100% Newlands) and Case 4 (100% Wara).

Table 1. NO formation/reduction model.

Model No.	Zeldovich NO	Prompt NO	Volatile fuel NO	Char fuel NO
Model 1			Eqs. (2.17)&(2.18)	
			(De Soete)	
Model 2	Eqs. (2.1)-(2.11)	Eqs. (2.12)-(2.16)	Eqs. (2.19)&(2.20)	Eqs. (2.26)&(2.27)
			(Chen et al.)	
Model 3			Eqs. (2.21)-(2.25)	
			(Mitchell et al.)	

Table 2. Summary of mathematical models used in simulations.

Phenomena	Mathematical model
Turbulence	RNG k-ε [44]
Thermal Radiation	Discrete Ordinate [45]
Devolatilization	Modified TDP model [43]
Gas phase combustion	Combined model of kinetics and eddy
	dissipation [46]
Char combustion	Field et al. [47]
Char combustion zone transition	Essenhigh et al. [48]
Particle's tracking	Lagrangian
Turbulence effect on particle motion	Stochastic [49]

Table 3. Coal properties.

Coal	Newlands	Wara
Proximate analysis [wt%]		
Moisture* (as-received)	2.5	41.2
Moisutre (after pulverized)	0.0	40.1
Volatile matter*	26.6	51.4
Fixed carbon*	58.0	46.5
Ash*	15.4	2.1
Fuel ratio [-]	2.18	0.90
Ultimate analysis* [wt%]		
C	69.6	67.0
Н	4.8	5.2
N	1.6	1.2
O	8.2	23.9
Combustible sulfur	0.41	0.63
Lower heating value (LHV)* [MJ/kg]	27.13	25.20
Mass fraction of particles [%]		
$D_p = 5 \mu \text{m}$	6.8	0.9
$D_p = 20 \mu \mathrm{m}$	26.8	23.2
$D_p = 40 \mu \text{m}$	24.0	22.8
$D_p = 60 \mu \text{m}$	16.3	11.4
$D_p = 80 \mu m$	12.2	10.6
$D_p = 100 \mu \text{m}$	14.0	31.2

*: Dry basis

Table 4. Cases investigated in this study.

Cara Na	Coal feeding rate ratio on LHV basis (%)		
Case No.	Newlands coal	Wara coal	
Case 1	100	0	
Case 2	75	25	
Case 3	25	75	
Case 4	0	100	